TAPERED ROLLER BEARING CAGES

STAMPED-STEEL CAGES

The most common type of cage used for tapered roller bearings is the stamped-steel cage. These cages are mass produced from low-carbon sheet steel using a series of cutting, forming and punching operations. These cages can be used in high temperature and harsh lubricant environments.



Fig. 8. Stamped-steel cage.

POLYMER CAGES

Cages for tapered roller bearings made of polymer material are used primarily for pre-greased and sealed package designs. The most common polymer materials used are Nylon thermoplastics with glass reinforcement. Polymer cages can be mass produced in large quantities and offer more design flexibility than stampedsteel types. Polymer cages are lightweight and easy to assemble. In some instances, increased bearing rating can be achieved by allowing one or two extra rollers in the bearing complement. Care should be exercised when using aggressive lubricants with EP (extreme-pressure) additives in combination with elevated temperatures greater than 107° C (225° F).

MACHINED CAGES

Machined cages for tapered roller bearings are robust in design and are suited for high-speed and high-load applications. Machined cages use alloy steels and are produced through milling and broaching operations. Assembly does not require a close-in operation and rollers can be retained using nibs or staking. Oil holes also can be easily added for extra lubrication for demanding applications. Some designs are silver plated for special applications.

PIN-TYPE CAGES

Tapered roller bearing pin-type cages retain the rolling elements by the use of a pin located through an axial hole in the center of the roller. Pin-type cages for tapered roller bearings consist of two rings with roller pins attached by screw threads at one end and welding at the other end. These types of cages are primarily used for larger tapered roller bearing designs (greater than 400 mm [15.7480 in.] O.D.). Pin-type cages are machined out of steel and typically allow for an increased number of rolling elements. Pin-type cages are restricted to low-speed applications (less than 20 m/sec [4000 ft/min] rib speed).

OPERATING TEMPERATURES

Bearings operate in a wide range of applications and environments. In most cases, bearing operating temperature is not an issue. Some applications, however, operate at extreme speeds or in extreme temperature environments. In these cases, care must be taken not to exceed the temperature limits of the bearing. Minimum temperature limits are primarily based on lubricant capability. Maximum temperature limits are most often based on material and/or lubricant constraints, but also may be based on accuracy requirements of the equipment that the bearings are built into. These constraints/limitations are discussed below.

BEARING MATERIAL LIMITATIONS

Standard bearing steels with a standard heat treatment cannot maintain a minimum hardness of 58 HRC much above 120° C (250° F).

Dimensional stability of Timken bearings is managed through the proper selection of an appropriate heat-treat process. Standard Timken tapered roller and ball bearings are dimensionally stabilized from -54° C (-65° F) up to 120° C (250° F), while standard spherical roller bearings are dimensionally stabilized up to 200° C (392° F) and standard cylindrical roller bearings are stabilized up to 150° C (302° F). Upon request, these bearings can be ordered to higher levels of stability as listed below. These designations are in agreement with DIN Standard 623.

TABLE 33.

Stability	Maximum Operating Temperature			
Designation	°C	°F		
SO	150	302		
S 1	200	392		
S2	250	482		
S 3	300	572		
S4	350	662		

With dimensionally stabilized product, there still may be some changes in dimensions during service as a result of microstructural transformations. These transformations include the continued tempering of martensite and decomposition of retained austenite. The magnitude of change depends on the operating temperature, the time at temperature and the composition and heat-treatment of the steel.

Temperatures exceeding the limits shown in table 33 require special high-temperature steel. Consult your Timken engineer for availability of specific part numbers for non-standard heat stability or high-temperature steel grades.

Suggested materials for use in balls, rings and rollers at various operating temperatures are listed in table 34. Also listed are chemical composition recommendations, hardness recommendations and dimensional stability information.

Operating temperature affects lubricant film thickness and setting, both of which directly influence bearing life. Extremely high temperatures can result in a reduced film thickness that can lead to asperity contact between contacting surfaces.

Operating temperature also can affect performance of cages, seals and shields, which in turn can affect bearing performance. Materials for these components and their operating temperature ranges are shown in table 35.

LUBRICATION LIMITATIONS

Starting torque in grease-lubricated applications typically increases significantly at cold temperatures. Starting torque is not primarily a function of the consistency or channel properties of the grease. Most often, it is a function of the rheological properties of the grease.

The high-temperature limit for greases is generally a function of the thermal and oxidation stability of the base oil in the grease and the effectiveness of the oxidation inhibitors.

See the LUBRICATION AND SEALS section on page 55 for more information on lubrication limitations.

EQUIPMENT REQUIREMENTS

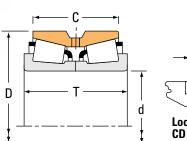
The equipment designer must evaluate the effects of temperature on the performance of the equipment being designed. Precision machine tool spindles, for example, can be very sensitive to thermal expansions. For some spindles, it is important that the temperature rise over ambient be held to 20° C to 35° C (36° F to 45° F).

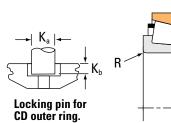
Most industrial equipment can operate at considerably higher temperatures. Thermal ratings on gear drives, for example, are based on 93° C (200° F). Equipment such as gas turbines operates continuously at temperatures above 100° C (212° F). Running at high temperatures for extended periods of time, however, may affect shaft and housing fits, if the shaft and housing are not machined and heat-treated properly.

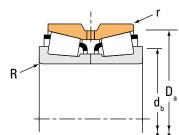
Although bearings can operate satisfactorily up to 120° C (250° F),

TYPE TNA









	Bearing D	imensions									
			Double Outer Ring	Load Ratings							
Bore d	0.D. D	Width T	Width C	Dynamic ⁽¹⁾		Factors ⁽²⁾			Dynamic ⁽³⁾		Factors ⁽²⁾
u	D	'		C ₁₍₂₎	е	Y_1	Y ₂	C ₉₀	C_{a90}	$C_{90(2)}$	K
mm in.	mm in.	mm in.	mm in.	N lbf				N Ibf	N Ibf	N lbf	
101.600 4.0000	190.500 7.5000	127.000 5.0000	101.600 4.0000	797000 179000	0.33	2.02	3.00	119000 26700	68000 15300	207000 46400	1.74
101.600 4.0000	190.500 7.5000	127.000 5.0000	104.775 4.1250	929000 209000	0.33	2.02	3.00	138000 31100	79300 17800	241000 54200	1.74
104.775 4.1250	180.975 7.1250	104.775 4.1250	85.725 3.3750	603000 135000	0.39	1.75	2.61	89700 20200	59200 13300	156000 35100	1.51
114.300 4.5000	190.500 7.5000	106.362 4.1875	80.962 3.1875	633000 142000	0.42	1.62	2.42	94300 21200	67100 15100	164000 36900	1.40
114.300 4.5000	228.600 9.0000	115.888 4.5625	84.138 3.3125	655000 147000	0.74	0.92	1.36	97500 21900	123000 27600	170000 38200	0.79
114.300 4.5000	228.600 9.0000	115.888 4.5625	84.138 3.3125	862000 194000	0.74	0.92	1.36	128000 28800	162000 36400	223000 50200	0.79
127.000 5.0000	182.562 7.1875	85.725 3.3750	73.025 2.8750	466000 105000	0.31	2.21	3.29	69400 15600	36300 8160	121000 27200	1.91
133.350 5.2500	190.500 7.5000	85.725 3.3750	73.025 2.8750	492000 111000	0.32	2.10	3.13	73300 16500	40300 9060	128000 28700	1.82
142.875 5.6250	200.025 7.8750	93.665 3.6876	73.025 2.8750	499000 112000	0.34	2.01	2.99	74300 16700	42800 9610	129000 29100	1.74
146.050 5.7500	236.538 9.3125	131.762 5.1875	106.362 4.1875	897000 202000	0.44	1.53	2.27	134000 30000	101000 22700	232000 52300	1.32
146.050 5.7500	241.300 9.5000	131.762 5.1875	106.362 4.1875	1040000 234000	0.32	2.12	3.15	155000 34800	84500 19000	269000 60600	1.83
149.225 5.8750	236.538 9.3125	131.762 5.1875	106.362 4.1875	897000 202000	0.44	1.53	2.27	134000 30000	101000 22700	232000 52300	1.32
149.225 5.8750	241.300 9.5000	131.762 5.1875	106.362 4.1875	1040000 234000	0.32	2.12	3.15	155000 34800	84500 19000	269000 60600	1.83
150.967 5.9436	229.873 9.0501	108.000 4.2520	116.000 4.5669	597000 134000	0.33	2.03	3.02	88900 20000	50600 11400	155000 34800	1.76
152.400 6.0000	254.000 10.0000	142.875 5.6250	111.125 4.3750	1150000 258000	0.41	1.66	2.47	171000 38500	119000 26800	298000 67000	1.43
165.100 6.5000	298.450 11.7500	142.875 5.6250	111.125 4.3750	1150000 258000	0.47	1.44	2.15	171000 38500	137000 30800	298000 67000	1.25
177.800 7.0000	282.575 11.1250	107.950 4.2500	79.375 3.1250	748000 168000	0.42	1.62	2.42	132000 29700	93900 21100	230000 51700	1.41

 $^{^{(1)}}$ Based on 1 x 10⁶ revolutions L_{10} life, for the ISO life-calculation method. $C_{1(2)}$ is the double-row radial value. $^{(2)}$ Consult your Timken engineer for instructions on use or review the Timken Engineering Manual on timken.com/catalogs. $^{(3)}$ Based on 90 x 10⁶ revolutions L_{10} life, for The Timken Company life-calculation method. C_{90} and C_{890} are radial and thrust values for a single row. $C_{90(2)}$ is the double-row radial value.

Part N	lumber		Dimensions							
		Shaft		Hou	ısing	Р	1			
Inner	Outer	Max Shaft Fillet Radius	Backing Shoulder Dia.	Max Housing Fillet Radius	Backing Shoulder Dia.			Bearing Weight		
		R ⁽⁴⁾	d_b	r ⁽⁴⁾	D_a	K_a	K_b			
		mm in.	mm in.	mm in.	mm in.	mm in.	mm in.	kg lbs.		
NA861	854D	3.5 0.14	120.0 4.72	1.5 0.06	174.0 6.85	- -	- -	14.98 33.03		
HH221449NA	HH221410D	3.5 0.14	122.0 4.80	1.5 0.06	179.0 7.05	- -	_ _ _	14.81 32.64		
NA782	774CD	3.5 0.14	122.0 4.80	1.5 0.06	168.0 6.61	19.05 0.75	7.13 0.28	10.20 22.50		
NA71450	71751D	3.5 0.14	132.0 5.20	1.5 0.06	181.0 7.13	- -	- -	11.08 24.43		
NA97450	97901D	3.5 0.14	140.0 5.51	2.3 0.09	213.0 8.38	- -	- -	19.81 43.70		
HM926740NA	HM926710CD	3.5 0.14	146.0 5.75	2.3 0.09	219.3 8.63	19.05 0.75	8.73 0.34	20.61 45.41		
NA48291	48220D	3.5 0.14	141.0 5.55	0.8 0.03	176.0 6.93	- -	_ _	7.13 15.71		
NA48385	48320D	3.5 0.14	148.0 5.83	0.8 0.03	184.0 7.24	- -	- -	7.45 16.45		
NA48686	48620D	3.5 0.14	158.0 6.22	0.8 0.03	193.0 7.60	- -	- -	8.41 18.56		
NA82576	82932D	3.5 0.14	166.0 6.54	1.5 0.06	226.0 8.90	- -	- -	20.60 45.41		
HM231140NA	HM231116D	3.5 0.14	164.0 6.46	1.5 0.06	224.0 8.82	- -	_ _	22.00 48.50		
NA82587	82932D	3.5 0.14	169.0 6.65	1.5 0.06	226.0 8.90	- -	_ _	19.82 43.73		
HM231149NA	HM231116D	3.5 0.14	167.0 6.57	1.5 0.06	224.0 8.82		-	21.24 46.82		
M231647	M231616XD	3.5 0.14	168.0 6.61	**	222.0 8.74	_ _	_ _ _	16.14 35.57		
NA99600	99102CD	3.5 0.14	177.0 6.97	1.5 0.06	238.0 9.37	0.87	0.31	26.54 58.51		
NA94650	94118D	3.5 0.14	190.0 7.48	1.5 0.06	272.0 10.71	_ _ _	_ _ _	41.69 91.93		
NA87700	87112D	3.5 0.14	200.0 7.87	1.5 0.06	267.0 10.50			23.05 50.80		

(4)These maximum fillet radii will be cleared by bearing corners.

NOTE: For desired fitting practice and resultant mounted setting range information, contact your Timken engineer.